

Development of a simulation scenario for cooperative robotics studies with marine crafts

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Abstract: - Marine environments offer suggestive scenarios for Automatic control and Cooperation strategies to be applied. The present paper focus on a particular one: Two ships towing together an off-shore oil retaining boom. Basic dynamical equations are presented for the combined displacement of both ships plus the boom. Computer simulation of basic manoeuvres works out the basic control implications of the problem and suggests cooperation among the ships as a suitable and reliable technique to fulfil ships goals and minimize boom strain.

Key-Words: - Unmanned marine vehicles, Behavior simulation, Adaptive control strategies, Decentralized control, Cooperative mobile robots, Coordination

1 Introduction

Concerning feedback control applied to marine vessels, Fossen [1] lists a wide number of examples of commercially available systems: Ship and underwater vehicle autopilots for course-keeping and turning control, way-point tracking trajectory and path control system for marine vessels, depth autopilots for underwater vehicles, torpedo control systems, attitude control system for underwater vehicles, dynamic positioning systems for marine vessels, positioning mooring systems for floating vessels, fin and rudder-roll stabilization systems, wave-induced vibration damping systems for high-speed craft, buoyancy control systems including trim and heel correction systems, propulsion control systems and forward speed control systems, propeller and thrusters control systems and energy and power management systems.

From the point of view of control problems; marine environments offer a wide source of suggestive scenarios:

- Ship's course control, rises as a first interesting problem: To determine the optimum course according to sea state, fuel consumption, time to delivery, etc. Navigation in shallow waters or along channels requires in both cases a sound strategy of course control to avoid run aground or to prevent collisions.
- Seakeeping problems [2], has been the scope of recent efforts: To improve ship's stability and global performance by means of proper actuators such as fins, T-foils or flaps [3], [4], [5]. Such

actuators require also some kind of control smartness to be operated properly.

- Manoeuvring is perhaps a still more varied field. It can include a single vessel or several ones in a wide range of scenarios:

The apparently simple operation of freight or person offshore transfer, involved a coordinated manoeuvre between two vessels. The degree of complexity depends on several factors; weather conditions, ships features, kind of freights to be transferred and human factors could be pointed out. Recent papers on related topics are [6], [7] and [8].

The mutual manoeuvring between two or more sailing vessels to avoid a possible collision, taking into account the operational constraints and course objectives, compose also a complex system. In this context, towing cases are of new interest [9].

Device deployment constitutes another interesting scenario of waterborne operation. The deployment of nets, set of buoys, barriers etc. which may be employed to mark or confine a particular sea area (for example, after an oil leakage). This scenario includes also the removal of wrecks and other recycling operations. Several watercrafts may cooperate to hold, transport and eventually deploy the device in a proper way.

Operational requirements in this case demand the capability of involved watercrafts to perform a proper dynamic positioning, the ability of them to deal with the device to be deployed in a well coordinated way and capability to react against possible modifications on the area to be bounded. The success demands the correct use and operation of the whole system as in the previously described operations.

It is clear, that thinking in terms of automation, the aforementioned scenarios are far beyond the scope of feedback control and emergent concepts or techniques such cooperation, interchange of information, multi-agent systems, self-awareness, reactive/deliberative answer capacity, etc. could play an interesting role to furnish a new generation of vessels able to perform such manoeuvres safer and better.

The need of cooperative control in marine operations has been recognized recently by several authors and institutions. [10] and [11] are illustrative references. A way to deal with this kind of problems is to look at the more general robotics field, where cooperation between mobile agents is attracting research interest from years ago. Reference books of interest are [12] and [13]; recent relevant articles focusing on formations and agent interaction are [14].

The present paper contains the first steps towards a complete study of a deployment scenario; two boats cooperate to deploy a floating barrier. The study departs from a simplified version of ships and barrier dynamic equations.

2 Problem Formulation

For marine vessels moving in 6 degrees of freedom (DOF), 6 independent coordinates are necessary to determine position and orientation.

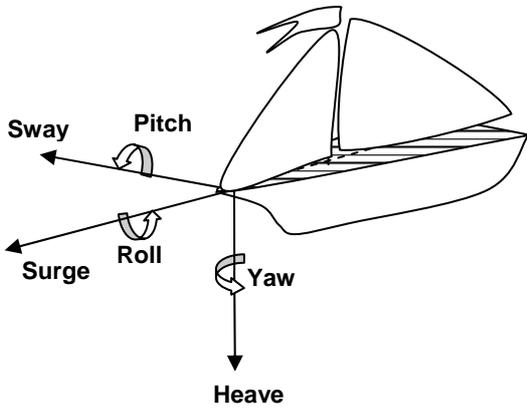


fig.1, Motion variables for marine vessels

Fig. 1 shows the 6 degrees of freedom as they are usually defined.

In the present work, only surge, sway and yaw are taken into account for being more directly related with ship course description.

2.1 Simplified scenario

The scenario is made up by two identical ships which tow together a floating boom. This last, consists of a

certain number of identical floating rigid elements. Two consecutive elements are jointed by a hinge, so that one can swing relative to the other. The whole set of elements form a sort of chain in which each rigid element acts as a link. Each one of the two tip links of the boom is jointed to the stern of one ship. Fig.2 shows a schematic view of the described scenario.

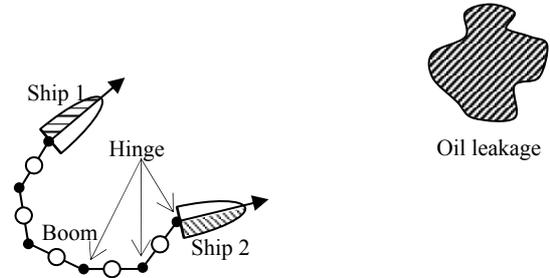


fig.2, Schematic scenario view

For ships and boom links, only their lengths will be considered as relevant for dynamical analysis.

Ships are described by their mass, mass inertia moment and three drag coefficients that represent the resistance to motion through the fluid along surge, sway and yaw coordinates. Propulsion in surge direction is considered in terms of power and steer action is also represented in terms of power to generate a moment in the yaw direction. The combined effect of propulsion power and steer power determines the ship course; both variables will be the input variables to be operated for control purposes.

Booms links are defined also by their mass, mass inertia moment and drag coefficients similar to those employed in the ships. Strains in the ends of the links complete the description for the boom dynamic.

2.2 Mathematical approach

The motion of a ship, in the x-y plane, can be described by mean of the following equations:

$$m_b a_{bx} = [F_m - \mu_l (v_{by} \sin(\theta) + v_{bx} \cos(\theta))] \cos(\theta) - \mu_t l_s (v_{bx} \sin(\theta) - v_{by} \cos(\theta)) \sin(\theta) \quad (1)$$

$$m_b a_{by} = [F_m - \mu_l (v_{by} \sin(\theta) + v_{bx} \cos(\theta))] \sin(\theta) - \mu_t l_s (-v_{bx} \sin(\theta) + v_{by} \cos(\theta)) \cos(\theta) \quad (2)$$

$$M - \mu_a l_s \omega_b = I_b \alpha_b \quad (3)$$

where m_b represents the mass of the ship, a_{bx} and a_{by} represent the acceleration in axes x and y, F_m is the surge force applied to the ships, M is the yaw

moment, μ_s , μ_t , μ_a are the drag coefficients in surge, sway and yaw, l_s is the length of the ship, I_b is the moment of inertia.

The moment M , applied in the yaw direction, drives the changes of course for the ship. In the present case, surge force and yaw moment have been decoupled and both act independently.

Figure 3 shows the geometry of the problem where the angle θ represents the course, v_b is the ship velocity and mcb is the position of the ship centre of mass.

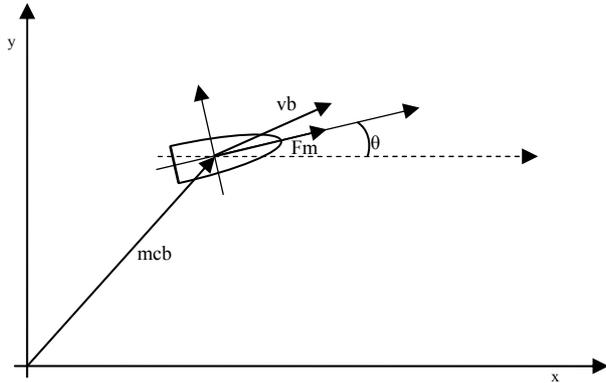


fig.3, Ship main variables

The motion of the boom, which is attached to the ships, is deduced by first considering a link and then combining several links. A closing condition is imposed to assure boom continuity.

Figure 4 shows the geometry of the boom. Were n_i represents a unit vector normal to a generic link, p_i is a unit vector parallel to the link r_i is the position of the link centre of mass and l represents half the length of the link.

The motion of a single generic link is given by:

$$\bar{T}_{i,i+1} - \bar{T}_{i-1,i} - \frac{(|\bar{v}_i \cdot \bar{n}_i|s + |\bar{v}_i \cdot (-\bar{p}_i)|q)}{|\bar{v}_i|} \bar{v}_i = m\bar{a}_i \quad (4)$$

were, T represents the strain in a tip of the link, q and s represent longitudinal and perpendicular drag coefficients and m is the mass of the link.

$$(\bar{T}_{i,i+1} \cdot \bar{n}_i)l + (\bar{T}_{i-1,i} \cdot \bar{n}_i)l - A\omega_i = I\alpha_i \quad (5)$$

were A is a drag coefficient, and I is the moment of inertia.

The closing condition is imposed by means of eq. (6)

$$\bar{r}_i - l\bar{p}_i - l\bar{p}_{i+1} - \bar{r}_{i+1} = 0 \quad (6)$$

were r_i is the position of the link.

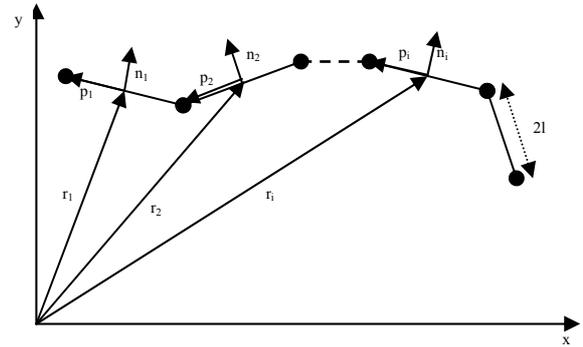


fig.4, Geometry of the boom

3 Scenario Studies

The research begins with the simplest case. Both ships try to go in parallel, along a straight path. Figure 5 shows the result for a boom composed of five elements, the arrows in the extremes represent the successive positions and orientation of both ships. The boom has been represented by lines with a circle in the centre of each element. The boom pulls the stern of the ships, and the ships rotate. Eventually, both ships make a tug of war. The need of a control action on the rudders, to counteract the boom tug, is clear.

Initial conditions have been highly forced. So, ships start their movement forming a square angle with the boom, which has been arranged all its length extended. This means that slight movements of the ships are going to generate large strains in the boom.

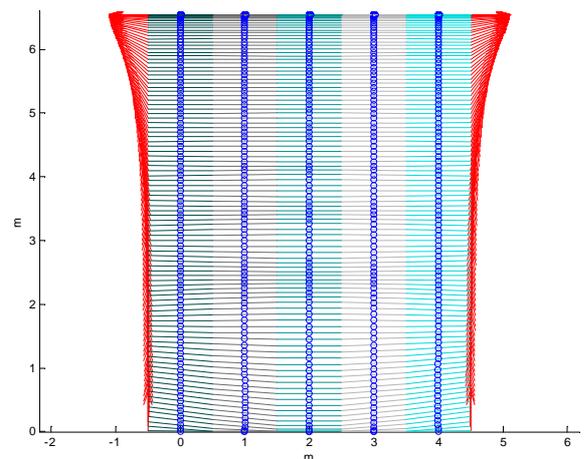


fig.5, Motion of the system without control

These initial conditions are hardly expectable in the real world, but they have been considered because represent an extreme case in which the ships and

boom dynamics are highly interrelated. The strain generated by the boom in its junctions with the aft of the ships tends to turn these outwards. As far as no effort has been applied by the ships to preserve their course, these begin to turn. This effect still increases more the boom's strain which increases again the rotation of the ships. Eventually, both point to opposite directions.

Now, a simple proportional control, acting over the yaw moment, is put into action to force the ships to follow a straight course. Figure 6 shows the simulated experiment.

A transient appears at the beginning, due to the tightness of the boom and then the ships tend to join smoothly, bending up the boom.

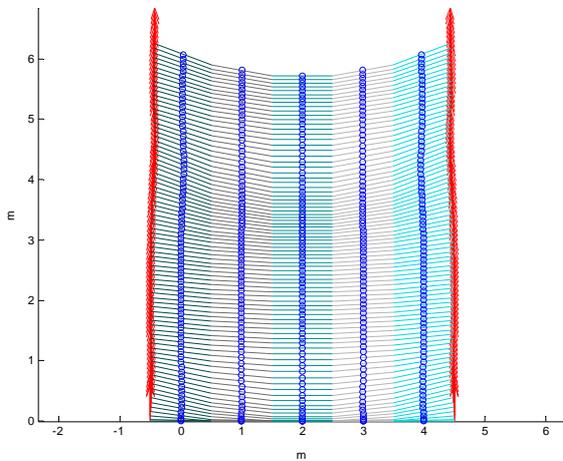


fig.6, Motion of the system with individual course control

Even though the course of the ships is now preserved, dragging the boom in such a way imposes the system to take unnecessary strain. To improve the control, a proportional control of the mutual distance has been added to the proportional course control in each ship. The idea is to establish a navigation distance between both ships.

The idea is to force both ships to come close, diminishing in this way the strain in the boom. Figure 7 shows the results obtained: both ships approach till they reach the set point distance, which has been established in four meters. Obviously, the system yields a better performance.

Anyway, this manoeuvre has still important shortcomings. Departing from the initial conditions prescribed for the model, in which the chain is so tight, the effort of both ships to approach one another increases still more the strain in the boom. A more intelligent manoeuvre would be that both ship change its course in the same direction, an after a while recover the original course. It is no difficult to realize that better solutions imply different manoeuvres for

each ship and therefore the necessity of coordination performed at an upper level to take decisions.

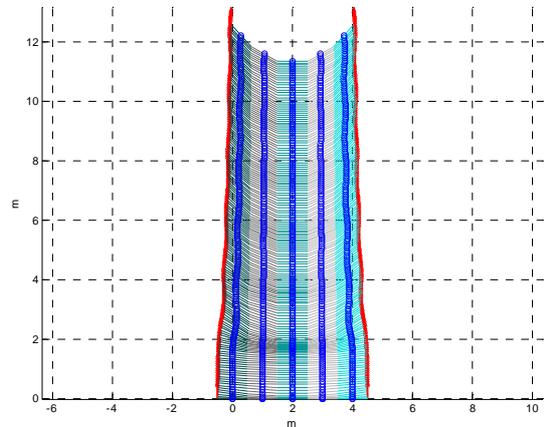


fig.7, Motion of the system with course and navigation distance control

Figure 8 compares the values of the horizontal component of the tension between the ship located in the left side and the last link of the boom at which the ship is connected, for the three previously described cases.

For the case on no course control, the tension increases at the same time that the ship rotates counter clockwise. Eventually, this tension will become equal to the force exerted by the ship in the surge direction.

In the case of course control, there are three different phases. First the course of the ships matches it set point. No control action is exerted but after a while ships begin to turn outwards, the control systems reacts and as a consequence the tension begin to rise. The combined effect on the tension plus the yaw moment exerted to amend the course causes that the ships were dragged by the boom inwards. The Tension reaches a maximum at some instant between 10 and 15 seconds, after this time, it begins to decrease due to the progressive approach of both ships and the bending of the boom.

The last case, when course and distance between ships is controlled, shows a sharper maximum than the previous one, located between 5 and 10 seconds. The reason is that the yaw moment acts from the beginning; as far as ships try to fulfil the set point impose to their mutual distance. Nevertheless, the tension falls after 15 seconds remaining for the rest of the period showed below the value of the tension for the previous describe case.

It also valuable to notice that there is still a remaining ripple in the last two cases studied, due to the strain exerted by the boom and the control effort that try to counteract this.

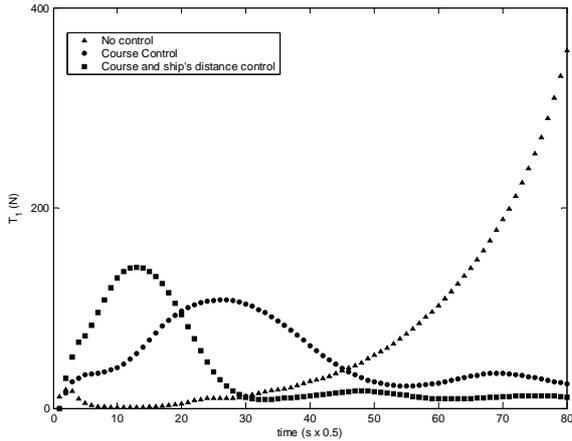


fig.8, Comparison among the tensions reached between the left side ship and the first link of the boom for three different cases: No course control is applied, individual course control is applied and course control and mutual distance between both ships is applied.

Tensions applied to the boom should be taking into account also before perform whatever manoeuvre. In the present study no limit has been imposed to these but it is quite obvious that there is a maximum limit for the strain that the boom can bear. After this limit the boom breaks. So, those manoeuvres that exceed this limit should be avoided.

Figure 9 shows also another interesting effect. It represents the angle between the ship direction (from aft to bow) and the ship velocity direction for both ships and for the two last cases discussed previously. In the first case, only course control applied, the left side ship presents always a positive angle and the right side ship presents a symmetrical situation. This means that during the whole period showed both ships are dragged inwards by the boom strain. This effect tends to diminish with time, as the strain of the boom is more and more relaxed.

In the second case, both ships sail nearer and the angle described begins to oscillate around zero. In this case, the reason for the difference of orientation between the ships and their velocities is due to the inertia of the ships; these change their course but the velocity remains still the same for a while, following a little bit delayed the new course of the ships.

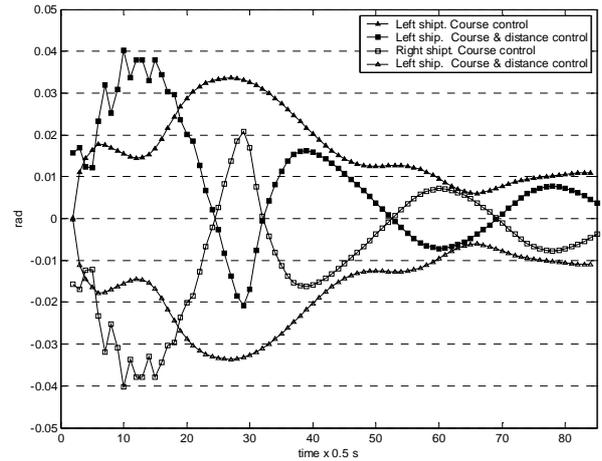


fig.9, Angle between ship direction and ship velocity direction

It is clear that for real cases, course changes must be studied. The simplest case is a single turn.

For this particular situation, the number of links in the boom has been increased till 25. In this way, the effect of the presence or absence of control action is clearly visible.

Figure 10 shows a 45° right turn. As can be seen in the figure, the outer ship must cover a longer distance to keep up a synchronous turn, if both ships have the same speed the outer ship delays, even causing that the outer ship cross over the boom.

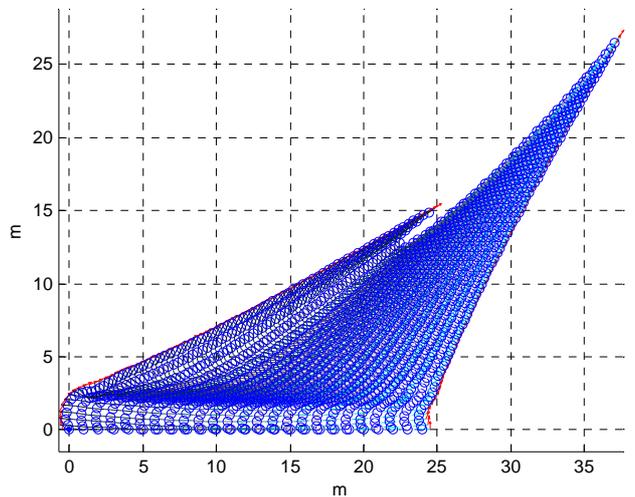


fig.10, System turn, ships with same speed.

A speed control is added, with a correction of the speed in function of the angle between the desired course and the line which links both ships. The speed of the ships is increased or decreased until the mentioned line and the desired course forms a 90° angle. Fig. 9 shows the effect of this control action. As can be seen, the outer ship, in this case the left

one, describes now a wider curve due to the increasing of its speed, compensating in this way the differences and remaining parallel to the right ship when the turn is finished.

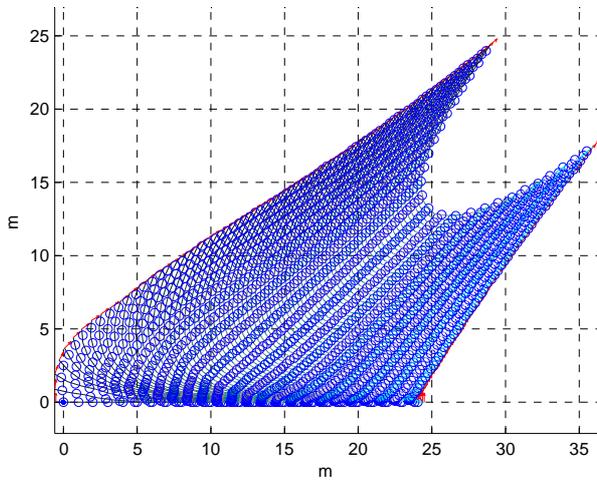


fig.11, System turns, with alignment control

4 Conclusion

The present paper is devoted to describe topic in the context of cooperative marine robotics.

Although the simplicity of the described scenario, suggestive situations arise after a careful insight.

In this paper, the problem was stated in mathematical terms, and a sequence of cases, from the simplest one, was studied. The various needs of control (coordinated) interventions appeared, and, for the moment, were toughly solved. Indeed, a future research will focus on better control.

The scenario is clearly suggestive, not only from the control point of view, but also from the cooperative strategies perspective.

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